

EXPERIENCE IN TURBOMACHINERY DESIGN

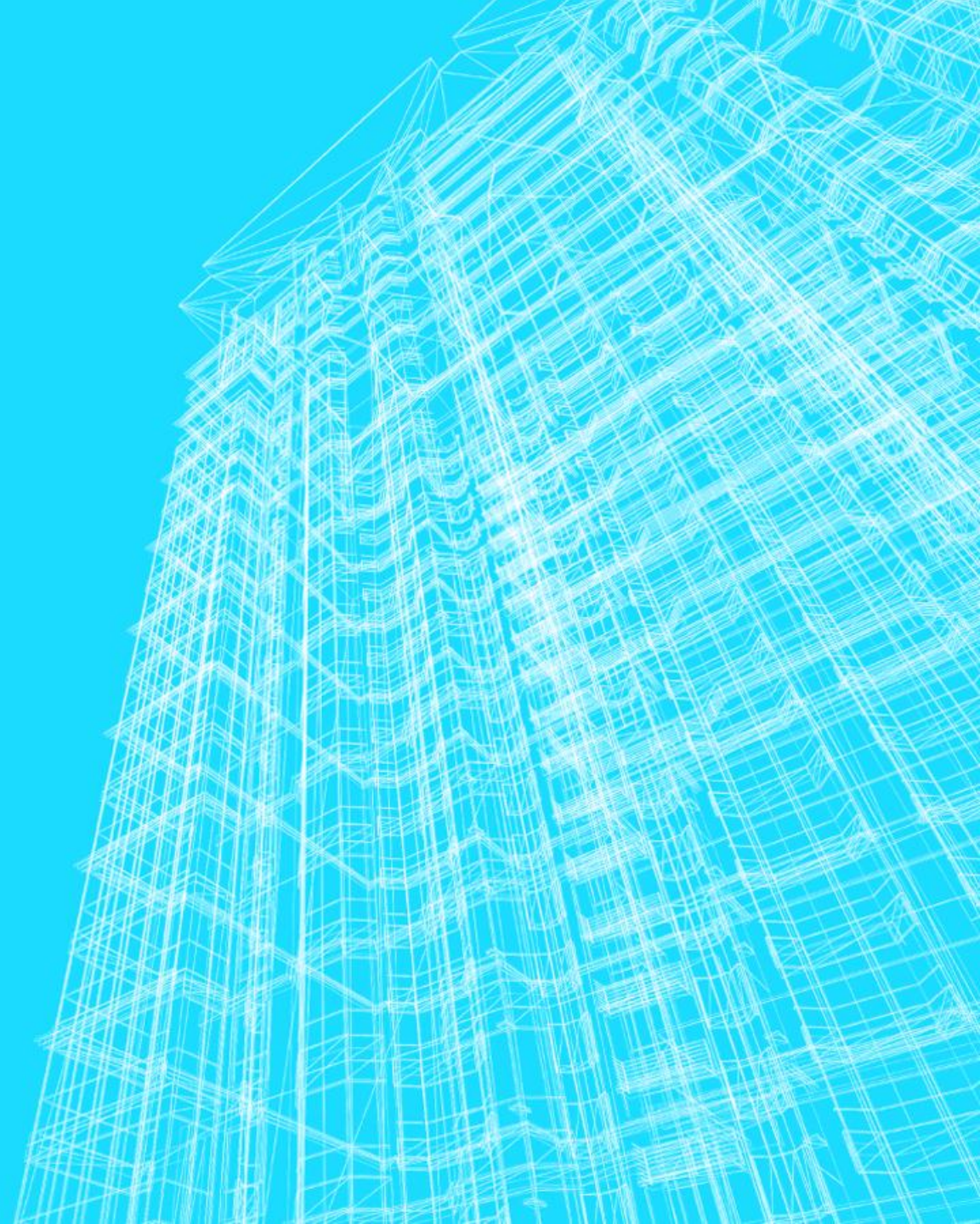
Cosimo Casotto

Ccasotto3@gatech.edu

+1 (510) 980-1940

Personal website: [cosimoLikesRockets](http://cosimoLikesRockets.com)

LinkedIn: [cosimo-casotto](https://www.linkedin.com/in/cosimo-casotto)



ROCKET ENGINE DESIGN PROJECT



Methane/Oxygen open gas generator
cycle liquid rocket engine

I found a YouTube tutorial on the use of **TURBOdesign1** last semester and I thought it would be fun to do a project with it. By the time I had access to the software I had been assigned a project for my rocket propulsion class. The **goal of the project** was to design the engine of a **single-stage-to-orbit (SSTO) rocket**. I used TURBOdesign1 to design the **powerpack** of the engine. If you are interested, please check out my full project report on my website.

[Rocket Propulsion / coding Projects | cosimoLikesRockets](#)

ROCKET ENGINE PARAMETER SUMMARY

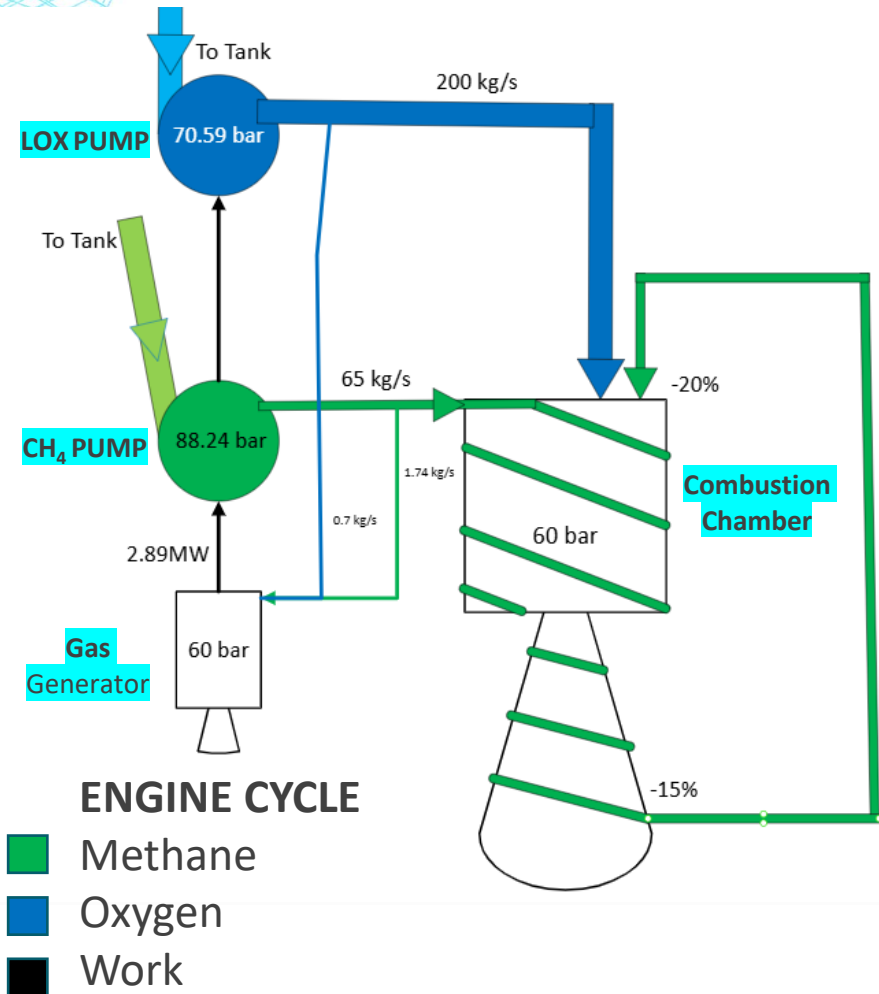
Inputs	Value
Design Thrust	1MN
Pcc	60 bar
m_dot (oxygen)	200 kg/s
m_dot (Methane)	65 kg/s
Design altitude	0 m (sl)
Number of Engines	1 Engine
Turn Start	130 km
Turn end	162.9 km

Outputs	Value
Final Altitude	162.9 km
Initial Mass	2909.03 kg
Nozzle Expansion ratio	30.5
Throat Radius	16 cm
Tcc	3522.98 K
Isp	384.67 s

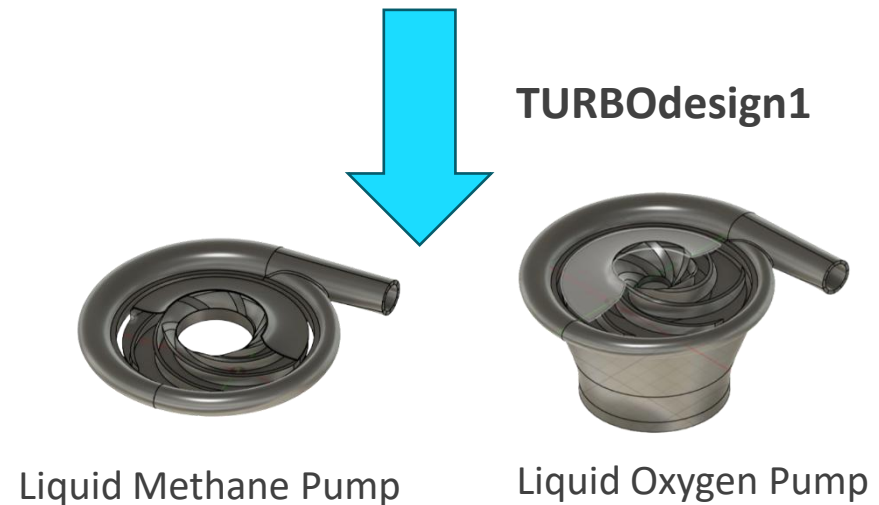
ENGINE CYCLE AND POWERPACK

I calculated the **pump parameters** using equations seen in class.

I used those calculations to plug the numbers into the TURBOdesign1 software to obtain performance plots and **blade profiles** of the oxygen and methane pumps.



	Oxygen Pump	Methane Pump
Head H [m]	599.11	2049.2
Flow Rate Q [m ³ /s]	0.17	0.15
Rotational Speed N [rpm]	6148.16	16470.16
Impeller Diameter D [m]	0.31	0.21
Specific Speed S_s	4.48	6.39
Power Input [MW]	1.37	1.52



PUMP CALCULATIONS

I used these equations to estimate the pump parameters

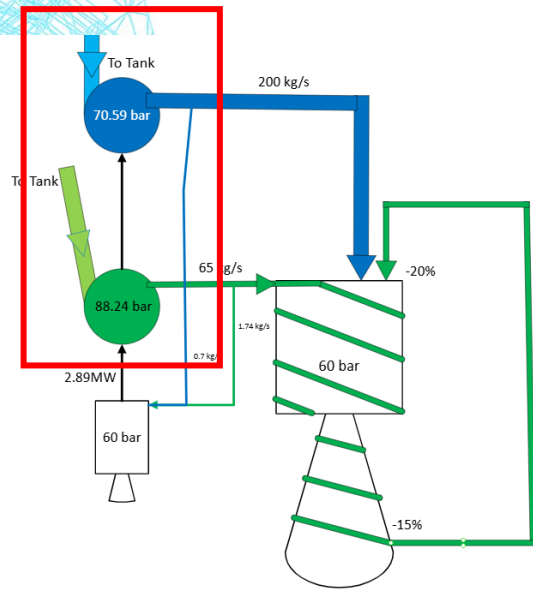
1. $H = \frac{\Delta P}{\rho g}$ Pump Head
2. $Q = \frac{\dot{m}}{\rho}$ Volumetric flow rate
3. $U_{\text{energy}} = \sqrt{\frac{g H}{\psi}}$ Tip velocity based on **the Euler head relation** for centrifugal pumps
4. $N = N_s \frac{(H g)^{0.75}}{Q^{0.5}}$ Rotational speed from specific speed
5. $D = d_s \frac{Q^{0.5}}{(H g)^{0.25}}$ Impeller diameter from specific diameter

6. $U_{\text{geom}} = \frac{D}{2} N$ Recalculate tip speed, based on the impeller diameter
 7. $\Psi_{\text{new}} = \frac{H g}{U_{\text{geom}}^2}$ Recalculate loading coefficient based on geometric tip speed → Accept if within **5%** of original value
 8. $P_{\text{hyd}} = \frac{\dot{m} \Delta P}{\eta \rho}$ Get pump hydraulic power
 9. $\text{NPSH} = \frac{P_{\text{in}} - P_{\text{vap}}}{\rho g}$
 10. $S_s = \frac{N Q^{0.5}}{(\text{NPSH } g)^{0.75}}$
- Cavitation check
- Accept if **3.7 < Ss < 9.1**

PUMP HYDRAULIC POWER

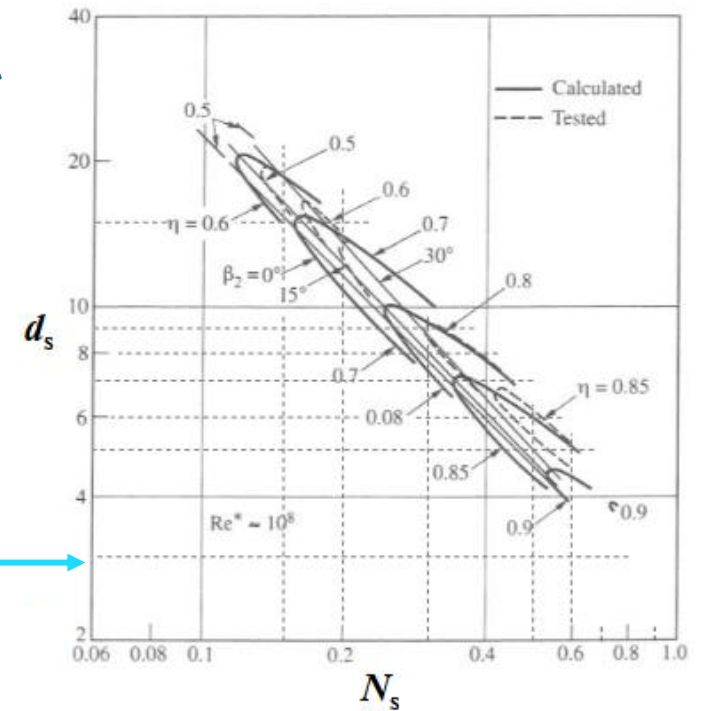
The **hydraulic power** of each of the two pumps is calculated as:

$$P_{hyd} = f(\dots)$$



$\psi = 0.59$
(between
0.4 and 0.7 for LREs)

From **Balje/Stepanoff**
similarity charts



Density from
thermodynamic state

Mass flow chosen for engine
performance parameters

$$f(\rho, \dot{m}, \Delta P, \psi, N_s, d_s, \eta, g, P_{in}, P_{vap})$$

Vapor pressure

3 bar (assumed tank pressure)

$\eta = 0.86$ arbitrarily chosen

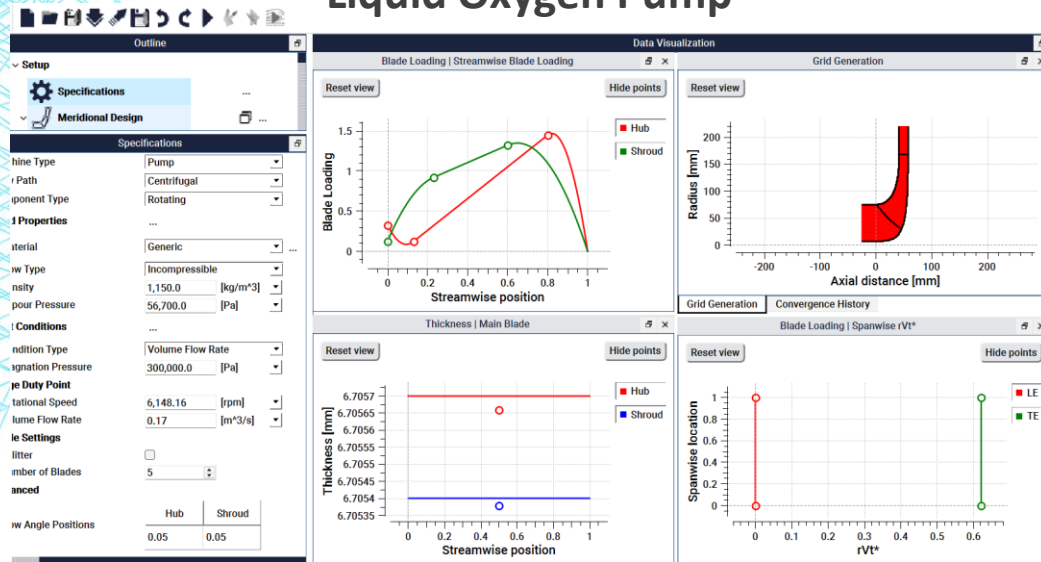
$\Delta P(\text{CH}_4) = \text{Pcc} / (0.85 * 0.8)$
assuming 20% loss on the injector
and 15% loss in the cooling
channels

$\Delta P(\text{LOX}) = \text{Pcc} / 0.85$ assuming 20%
loss on the injector

$$W_{total} = P_{hyd}(\text{LOX}) + P_{hyd}(\text{CH}_4)$$

PUMP PLOTS

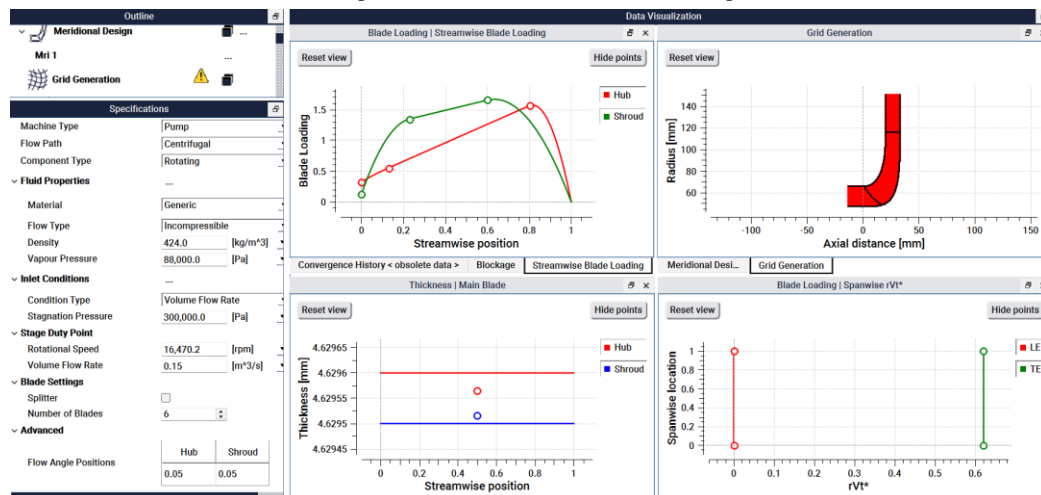
Liquid Oxygen Pump



Used **TURBOdesign1** to create the following plots:

1. Top Left: Streamwise Position vs **Blade Loading**. In both cases the hub is aft-loaded. There is a **risk of separation**. The loading on the shroud looks acceptable.
2. Top right: **Meridional design**.
3. Bottom left: Streamwise position vs **thickness**. I chose a constant thickness blade design.
4. Bottom right: **Swirl (rV_θ)** vs spanwise location. These plots look good, with 0 swirl at the inlet and a **constant swirl** at the trailing edge.

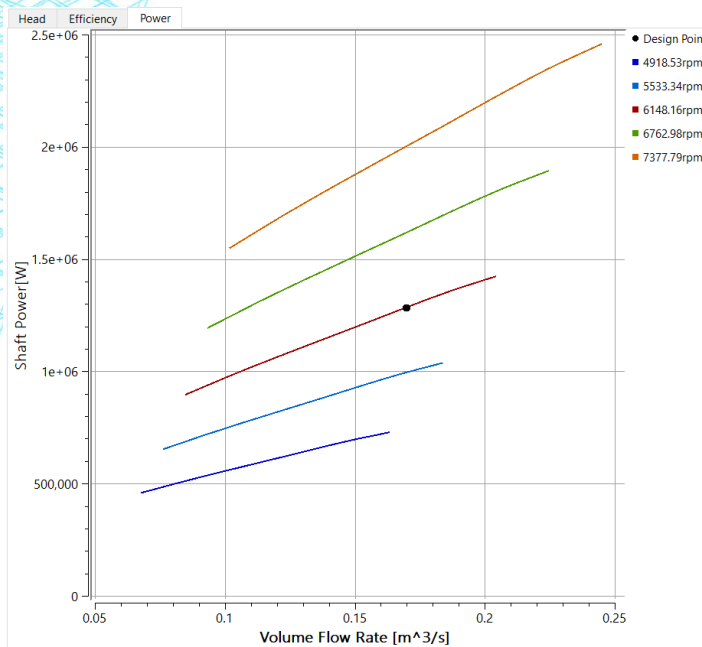
Liquid Methane Pump



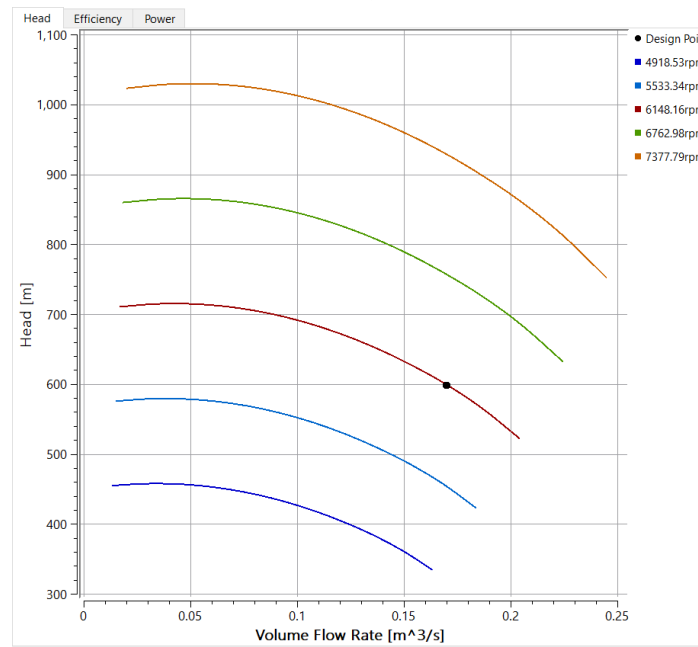
RESULTING PUMP PARAMETERS

TURBOdesign1 also generated these performance maps, which highlight the operating point of my oxygen pump.

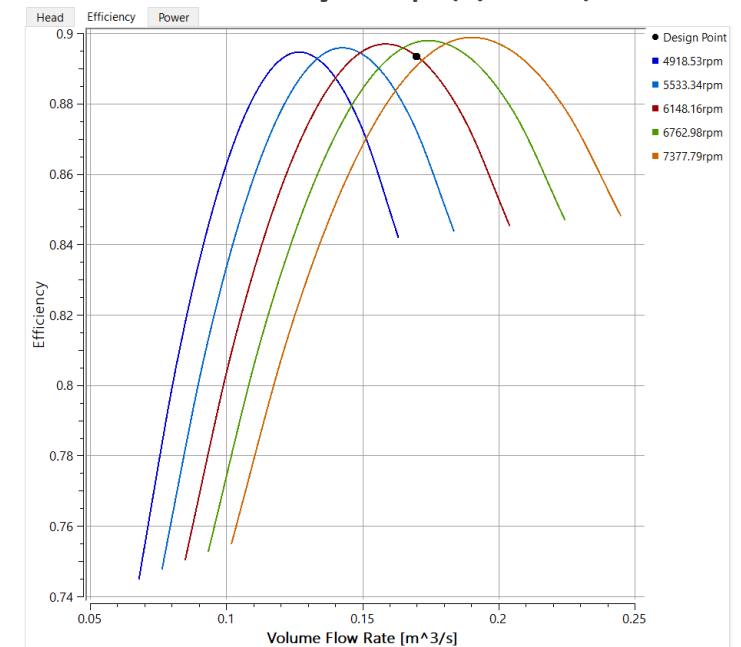
Power map (P_{shaft} vs Q)



Head map (H vs Q)



Efficiency map (η vs Q)



We can see that the oxygen pump is operating near the **Best Efficiency Point**. Slightly decreasing the volumetric flow rate would increase the efficiency and pump head while also decreasing the necessary shaft power.

However, we are already operating at a **higher efficiency** than the conservatively estimated efficiency ($\eta = 0.86$) for the preliminary calculations.

PUMP CAD



TURBOdesign1 has the option of outputting a **CAD file** of the pump **blade** as well as the **volute**. I imported these into Fusion360 and added the shroud and hub.

Note: This is only for the purpose of 3D printing the engine, the design of the **hub and shroud** is **not detailed**. Also, in the current version, both pumps are placed on the same shaft. Since they rotate at different speeds this is not an ideal design. Separating the shafts and using a gear system would improve the design.



LOX Pump



CH₄ Pump

The diagram illustrates a gas turbine cycle. On the left, a schematic shows air entering a compressor at 1.74 kg/s, being compressed to 60 bar, and then entering a combustion chamber where fuel is added at 200 kg/s. The combustion gases expand through a turbine, which produces work, and then exit through a nozzle. The turbine inlet temperature is 1200 K, and the exit temperature is 800 K. The nozzle exit temperature is 600 K. The cycle efficiency is 0.7.

On the right, the performance equations are given:

$$\frac{T_{out}}{T_{in}} = \left(\frac{P_{exit}}{P_{cc}} \right)^{\frac{\gamma-1}{\gamma}}$$

where T_{in} is the inlet temperature, T_{out} is the outlet temperature, P_{cc} is the combustion chamber pressure, and P_{exit} is the exit pressure. The efficiency is given by:

$$w_t = \eta_t c_p (T_{in} - T_{out}) = \eta_t c_p T_{in} \left[1 - \left(\frac{P_{exit}}{P_{cc}} \right)^{\frac{\gamma-1}{\gamma}} \right]$$

where w_t is the mass specific turbine work, η_t is the turbine efficiency, and c_p is the specific heat at constant pressure.

Combustion
 Gases
 $w_t = \eta_t c_p (T_{\text{in}} - T_{\text{out}}) = \eta_t c_p T_{\text{in}} \left(1 - \frac{T_{\text{out}}}{T_{\text{in}}}\right)$
 efficiency = 0.7

Mass flow
through the
turbine

Total power necessary
to run the pumps

$$\dot{m}_t = \frac{W_{\text{total}}}{w_t} = \frac{W_{\text{total}}}{\eta_t c_p T_{\text{in}} \left[1 - \left(\frac{P_{\text{exit}}}{P_{\text{cc}}} \right)^{\frac{\gamma-1}{\gamma}} \right]}$$

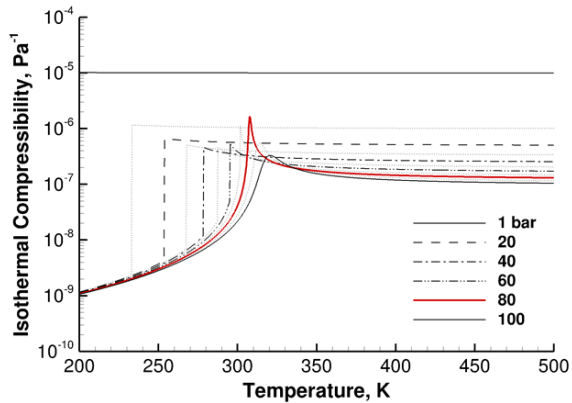
ABOUT MYSELF

- Graduating in **December** with a master's in Aerospace Engineering from **Georgia Tech**
- **CFD researcher** at Georgia Tech in Dr. Oefelein's High Performance Computing Lab
- Experience in **CFD** (Georgia Tech & NASA)
- Experience in **production engineering** (Rocket Factory Augsburg)
- Experience in **test engineering** (Technology for Propulsion and Innovation)



RESEARCH AT GEORGIA TECH

- My research centers around the **compressibility effects** in **supercritical mixing layers**
- In the vicinity of critical pressure and temperature, supercritical fluids exhibit **strong non-linearities** in their properties (ex: Isothermal compressibility plot in the bottom right of the slide.)
- The goal of my work is to identify and quantify the key **differences** between **ideal and supercritical** mixing, as well as in **single-species** and **multi-species** supercritical mixing

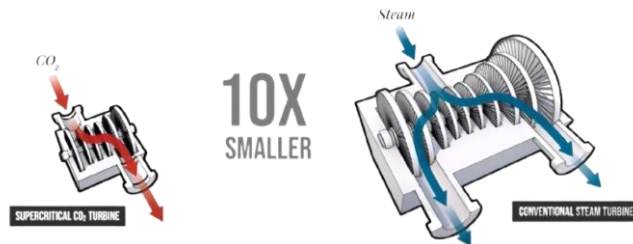


Isothermal compressibility

$$\alpha = \frac{1}{\rho} \left(\frac{\partial \rho}{\partial p} \right)_T$$

Supercritical fluids in turbomachinery:

- More efficient thermodynamic cycles
 - High heat capacity
 - No phase change, enabling higher temperatures
- Higher density
 - Smaller systems can produce the same amount of work



Setup of CFD simulations
for 2 different datasets

COLD CO_2

HOT CO_2

IDEAL gas
conditions

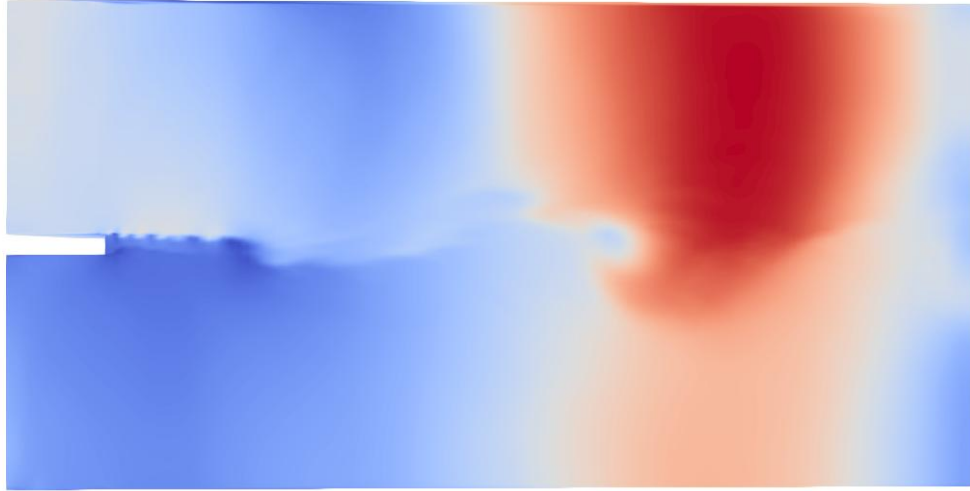
CH_4

O_2

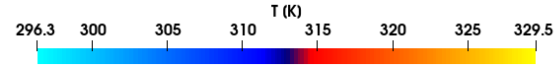
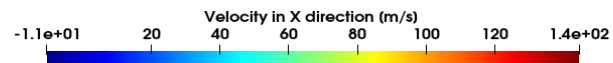
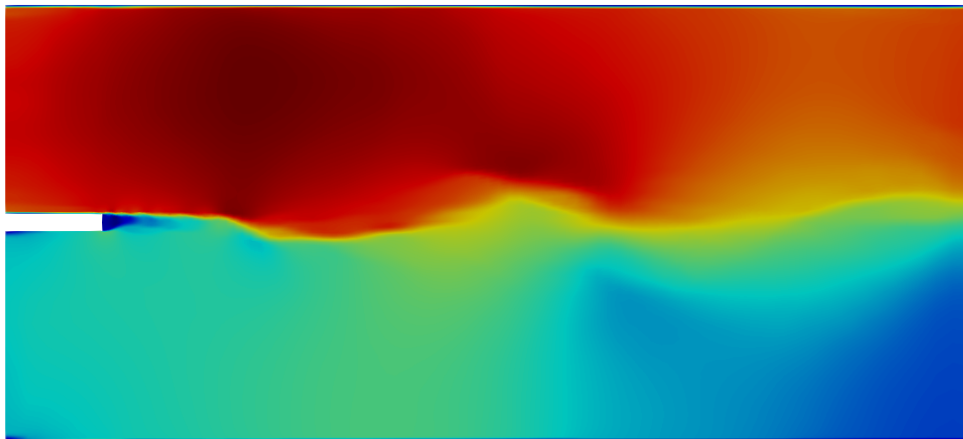
SUPERCritical
Multi-species mixing

SOME CONTOUR PLOTS

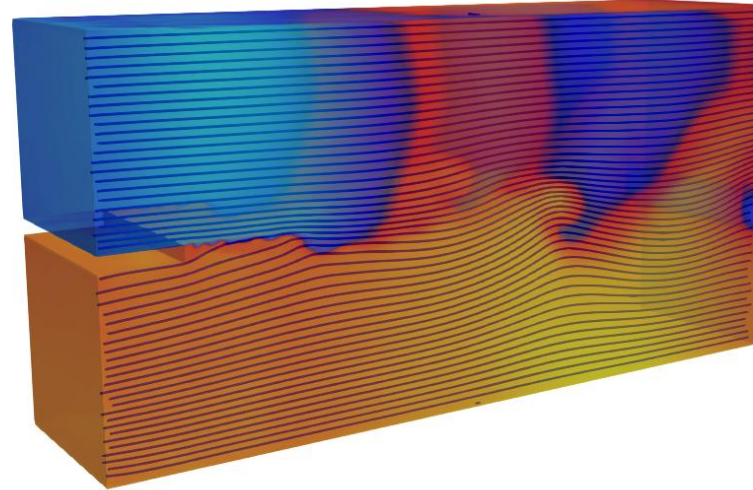
Density



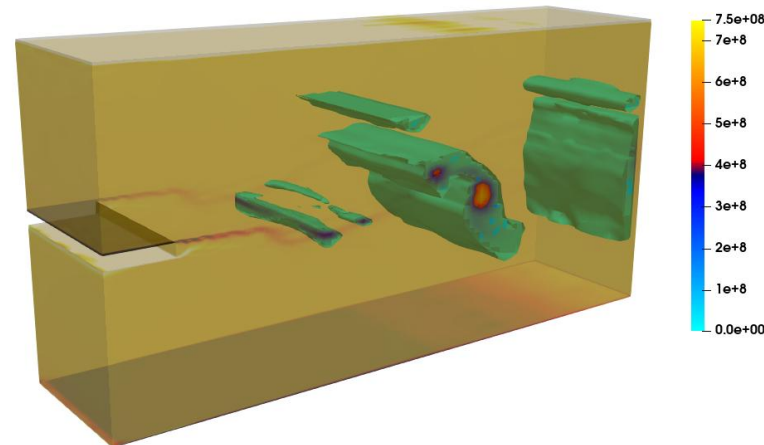
Velocity



Temperature



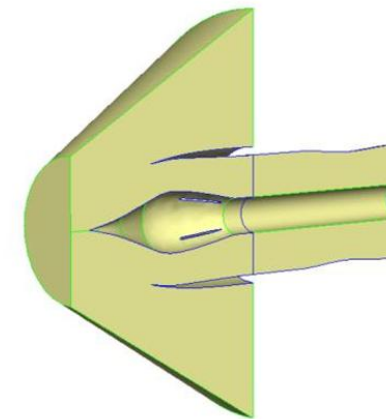
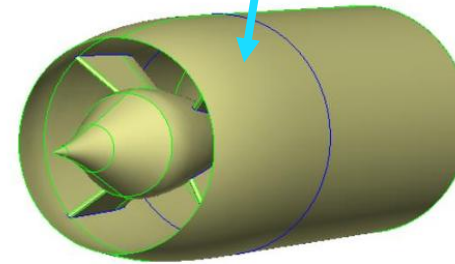
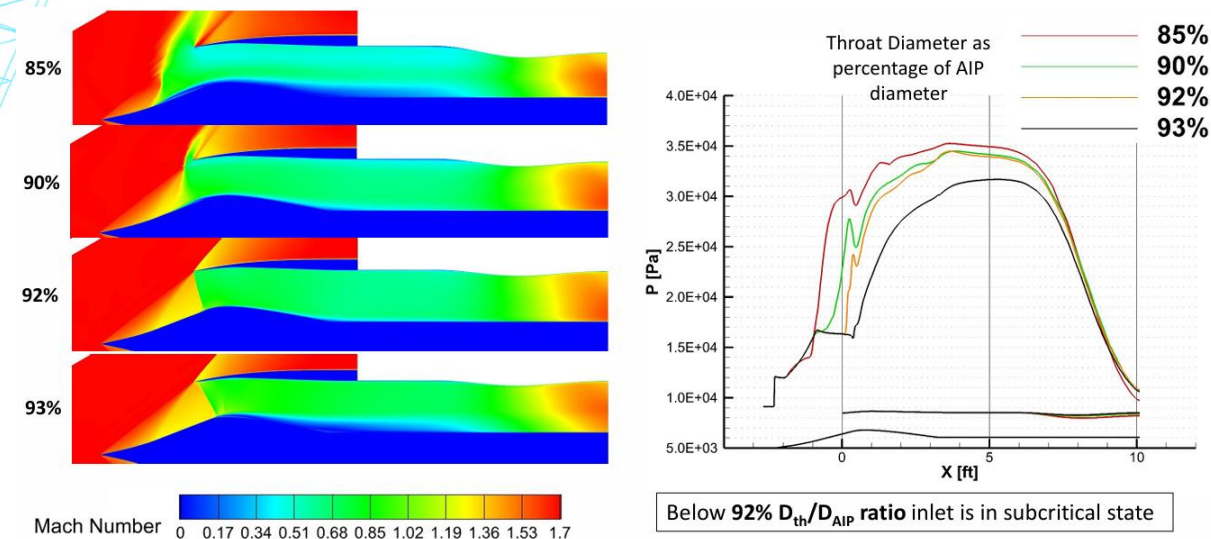
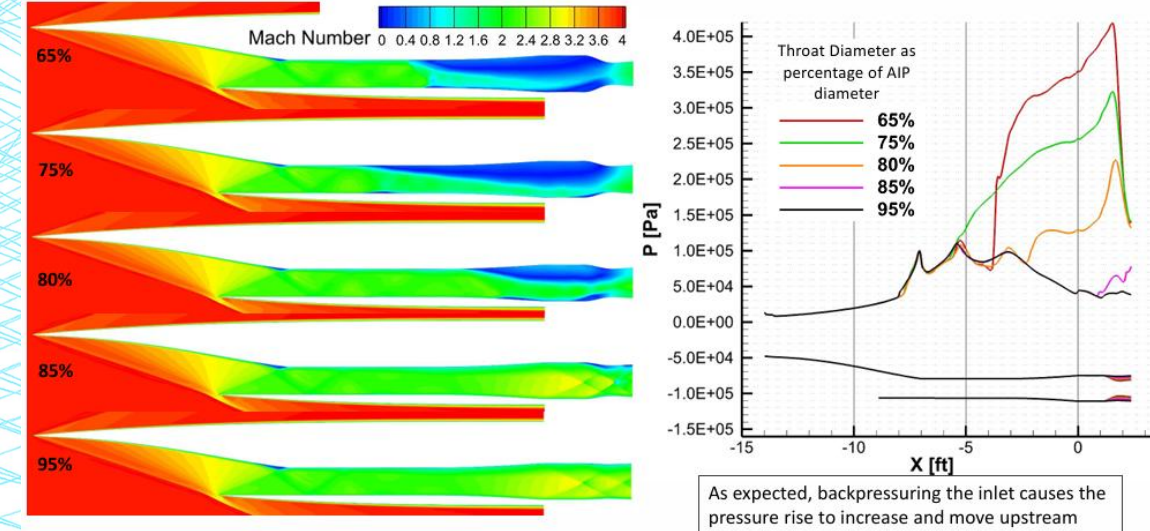
Q criterion



Two streams (upper/lower) mix downstream of the mixing plate. **Observations** on the **mixing behavior** of the 2 flows can be made.

CFD INTERNSHIP AT NASA

Wall pressure rise from increase in backpressure



Inlet negative-volume simulation domain

- Worked in the *Inlets and Nozzles* branch in the **Propulsion Division** at the **NASA Glenn Research Center**.
- Focused on creating and validating a **workflow** to speed up the **design and analysis** of supersonic **jet engine inlets**.
- Validated workflow for mixed-compression, Pitot, and streamline-traced inlets
- Demonstrated the **effects** of **backpressuring** the inlet by restricting the nozzle throat diameter.